

Ideas to create an adaptive finite element method for combustion problems

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A jointly work with:

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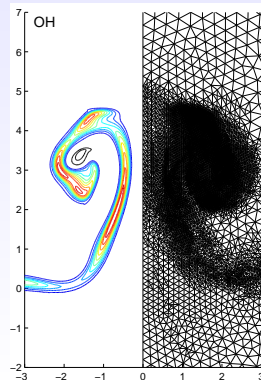
Amable Liñán

Students:

Pablo Fraile

Ángel Fernández

Workshop on mathematical modelling of combustion
23-25/05/2011



1 Physical phenomenon

2 Model I

- Mathematical model
- Numerical method
- Refinement strategies

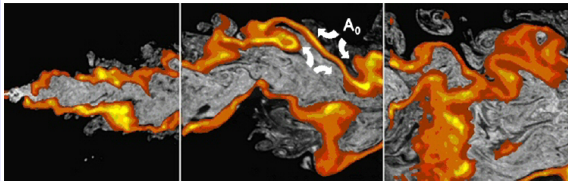
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3 Model II

- Mathematical model
- Numerical validation

4 Conclusions

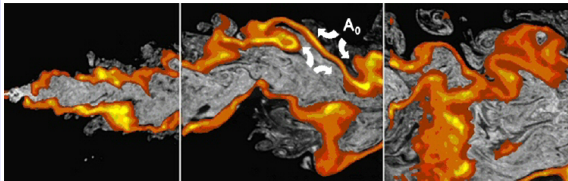
Physical phenomenon: Diffusion-Flame/Vortex Interactions



Rehm & Clemens (1997)

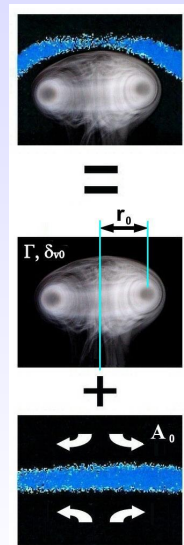
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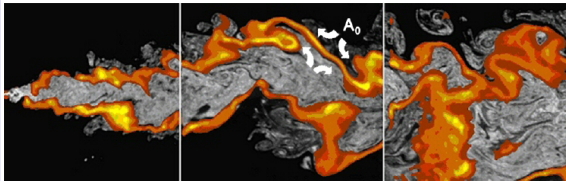


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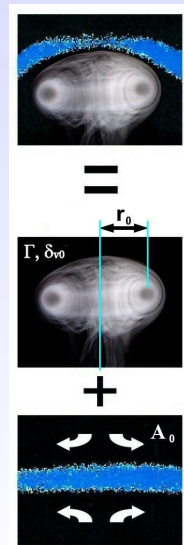


Physical phenomenon: Diffusion-Flame/Vortex Interactions



Rehm & Clemens (1997)

- Turbulent combustion occurs in the form of laminar flames embedded in thin mixing layers locally strained by vortices.
- A complete description of the phenomenon requires:
 - a good mathematical model.
 - an efficient numerical method.



Physical phenomenon: Conservation Equations

$$\begin{aligned}\frac{\partial \rho}{\partial \tau} + \nabla \cdot (\rho \mathbf{u}) &= 0 \\ \rho \left(\frac{\partial \mathbf{u}}{\partial \tau} + (\mathbf{u} \cdot \nabla) \mathbf{u} \right) &= -\nabla p + \frac{Pr}{Pe_0} \nabla \cdot [\mu (\nabla \mathbf{u} + \nabla \mathbf{u}^T)] \\ \rho \left(\frac{\partial h}{\partial \tau} + \mathbf{u} \cdot \nabla h \right) &= \frac{1}{Pe_0} \nabla \cdot \mathbf{q} \\ \rho \left(\frac{\partial Y_i}{\partial \tau} + \mathbf{u} \cdot \nabla Y_i \right) &= \frac{1}{Pe_0} \nabla \cdot \mathbf{j}_i + \mathcal{R} \dot{m}_i \quad i \neq N_2\end{aligned}$$

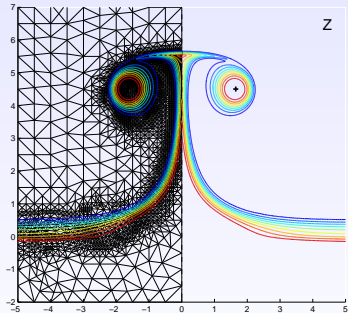
Scales

$$r_0, A_0, \rho_A, D_{TA}, h_A, \mu_A$$

Dimensionless parameters

$$Pe_0 = \frac{A_0 r_0^2}{D_{TA}}, \quad \mathcal{R} = \frac{A_{ext}}{A_0}, \quad \tilde{\Gamma} = \Gamma / (2A_0 r_0^2), \quad Pr = 0.72$$

- Model I (Simple model)
 - Constant fluid properties.
 - Infinitely fast chemical reaction.



Model I: Constitutive Equations

- Molecular transport models

$$\mathbf{j}_i = \rho D_i Y_i.$$

$$\mathbf{q} = -\lambda \nabla T + \sum_{i=1}^I h_i \mathbf{j}_i,$$

$$\text{with } \rho = \frac{\rho'}{\rho_A} = 1, \quad \mu = \frac{\mu'}{\mu_A} = 1, \quad \lambda = \frac{\lambda'}{\rho_A D_{TA} C_{pA}} = 1, \quad \rho D_i = \frac{\rho' D'_i}{\rho D_{TA}} = 1.$$

$$\text{and } h_i = h'_i / h_A, \quad h'_i = h'_{i,0} + C_{pA}(T' - T_0)$$

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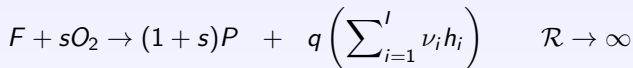
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- The chemical reaction is assumed to be infinitely fast:



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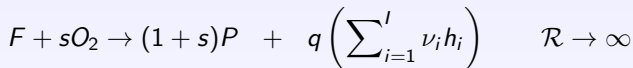
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- Energy and mass fraction equations introduce the mixture fraction Z defined as:

$$Z = \frac{SY_F/Y_{F,0} - Y_{O_2}/Y_{O_2,A} + 1}{1 + S} : \quad \left(\frac{\partial Z}{\partial \tau} + \mathbf{u} \cdot \nabla Z \right) = \frac{1}{Pe_0} \Delta Z$$

Model I: Original variables

- Z allows us to define the temperature, mass fractions of oxygen and fuel:

$$T = 1 + \gamma \left(\frac{1 - Z}{1 - Z_s} \right), \quad \frac{Y_F}{Y_{F,0}} = \frac{1 - Z}{1 - Z_s}, \quad Y_{O_2} = 0 \quad \text{if } Z \geq Z_s$$

$$T = 1 + \gamma \frac{Z}{Z_s}, \quad Y_F = 0, \quad \frac{Y_{O_2}}{Y_{O_2,A}} = 1 - \frac{Z}{Z_s} \quad \text{if } Z < Z_s$$

$$\text{with } Z_s = \frac{1}{S + 1}, \quad S = \frac{sY_{F,0}}{Y_{O_2,A}} \text{ and } \gamma = \frac{qY_{F,0}}{C_{pA}T_0(1 + S)}$$

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- The velocity field is given by the incompressible Navier-Stokes equation, which can be considered known for $Pe \gg 1$

$$u = \xi/2$$

$$v = -\eta$$

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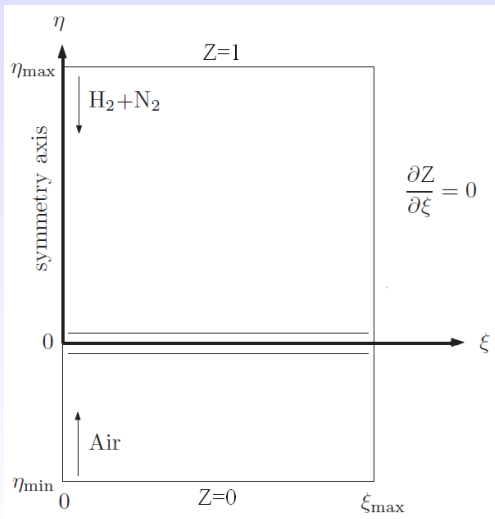
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$$u = \xi/2 - \frac{\tilde{\Gamma}}{\pi\xi_c} \left(\frac{\xi_c}{2\xi} \right)^{3/2} \left(\frac{\eta - \eta_c}{\xi_c} \right) I_1(\mu) \left[1 - e^{-(\rho/\delta_v)^2} \right]$$

$$v = -\eta + \frac{\tilde{\Gamma}}{\pi\xi_c} \left(\frac{\xi_c}{2\xi} \right)^{3/2} \left[\frac{\xi}{\xi_c} I_1(\mu) - I_0(\mu) \right] \left[1 - e^{-(\rho/\delta_v)^2} \right]$$

Hermanns et al., Combustion & Flame (2007)

Model I: Problem to solve



$$\left(\frac{\partial Z}{\partial \tau} + \mathbf{u} \cdot \nabla Z \right) = \frac{1}{Pe_0} \Delta Z$$

Initial condition:

$$Z_0 = 1 - \frac{1}{2} \operatorname{erfc} \left[(Pe_0/2)^{1/2} \eta \right]$$

Boundary condition:

Model I: Numerical method

- An efficient numerical treatment of multi-scales phenomena can be carried out with an adaptive method
- The idea is to adapt the domain of integration to the evolving features of the solution

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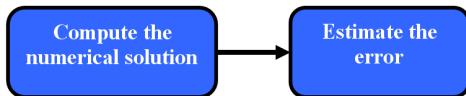
At time t_n

Compute the
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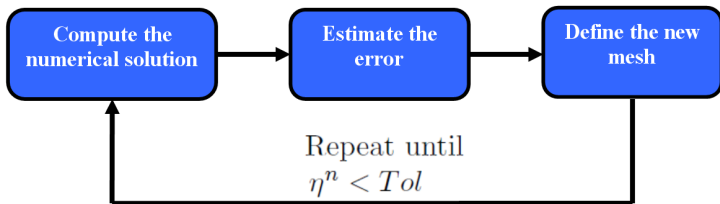
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At time t_n



Numerical Method: Compute the numerical solution

- To solve the problem we use a semi-Lagrangian scheme.

$$\left[\frac{\partial Z}{\partial \tau} + \mathbf{u} \cdot \nabla Z = \frac{1}{Pe_0} \Delta Z \right]_{(X(x, \tau_n; \tau), \tau)}$$

$X(x, \tau_n; \tau)$ characteristic curve backwards in time.

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$$\begin{cases} \frac{\partial \bar{Z}}{\partial \tau} = \frac{1}{Pe_0} \Delta \bar{Z} & \text{in } [X(x, \tau_n; \tau), \tau] \text{ with } \tau < \tau_n \\ \bar{Z}_h(x, \tau_{n-1}) & \text{initial condition} \end{cases}$$

where $\bar{Z}(x, \tau) = Z(X(x, \tau_n; \tau), \tau)$ and $X(x, \tau_n; \tau_n) = x$.

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- Solution:

- 1 Convection stage: The aim is calculate $\bar{Z}_h(x, \tau_{n-1})$.
 - Compute $X(x, \tau_n; \tau_{n-1})$ by Runge-Kutta(2).
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② Parabolic part: The aim is calculate $Z_h(x, \tau_n) = \bar{Z}_h(x, \tau_n)$.

- Crank-Nikolson scheme in time.
- Quadratic finite element in space.

Numerical Method: Error estimation I

Local a posteriori error will be computed.

- The error in the numerical integration is both in time and in space. However, $\Delta t = 0.005$ and we are only considering spatial adaptation.
- Spatial error can be evaluated as:

$$\eta_S^n = \eta_{conv}^n + \eta_{diff-reac}^n$$

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- For $Pe_0 \gg 1 \rightarrow \eta_{conv}^n \gg \eta_{diff-reac}^n$.
- Truncation error is evaluated in a new variable $W(x)$.

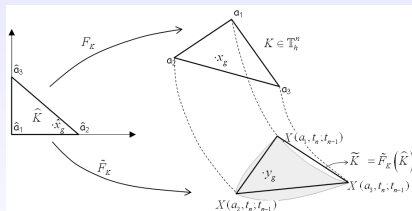
$$\eta^n = \left\| W_h^{n-1}(X(x, \tau_n; \tau_{n-1})) - \overline{W}_h^{n-1}(x) \right\|_{L_2}^2.$$

Numerical Method: Error estimation II

- For this problem the variable $W(x) = Z(x)$
- Truncation error:

$$\eta^n = \sum_K \left(\int_K \left[Z_h^{n-1}(X(x, \tau_n; \tau_{n-1})) - \bar{Z}_h^{n-1}(x) \right]^2 dK \right).$$

- We compute the integrals using quadrature points x_g .
 $x_g = F_K(\hat{x}_g)$ and $X^{n-1}(x_g) \approx y_g = \tilde{F}_K(\hat{x}_g)$.

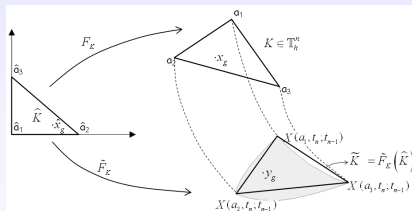


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- The optimal size A_K^{opt} to satisfy $\eta^n < Tol$ is

$$A_K^{opt} = A_K \left(\frac{Tol}{\sum_{K \in \mathbb{T}_h} \eta_K^{1/(\alpha+1)}} \right)^{\frac{1}{\alpha}} \eta_K^{-1/(\alpha+1)} \quad \text{with} \quad \alpha = 3$$

Isotropic Adaptation: Bisection

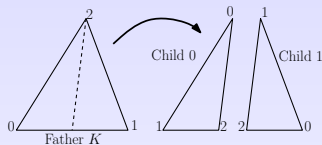
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- The element K (father) is divided into two elements (children) by cutting the refinement edge at its midpoint.

Bisection algorithm



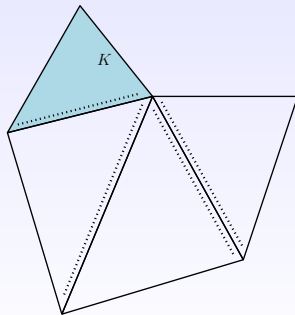
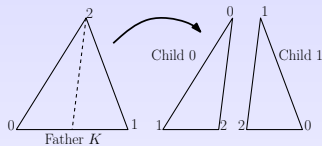
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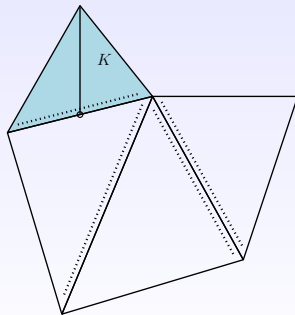
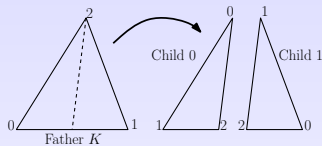
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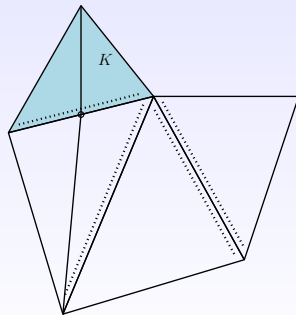
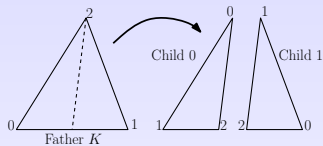
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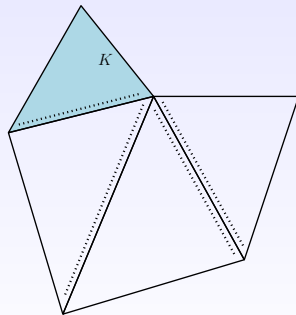
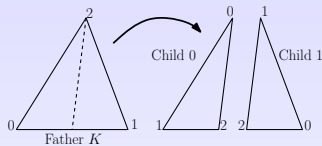
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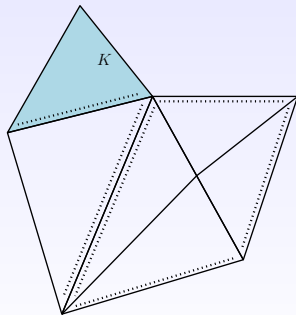
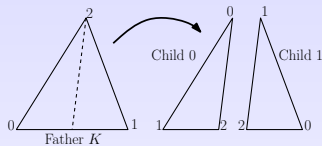
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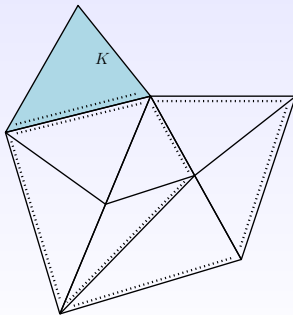
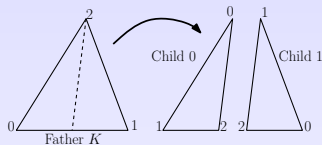
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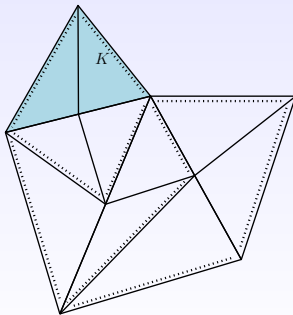
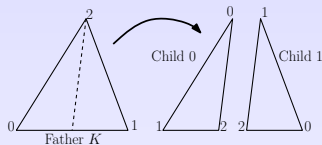
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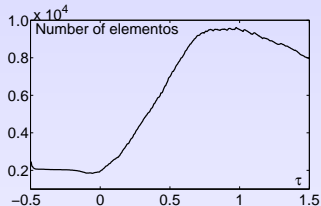


Isotropic Adaptation: 2D

- In 2D the refinement loop finishes if the refinement edge is the longest edge of each element.

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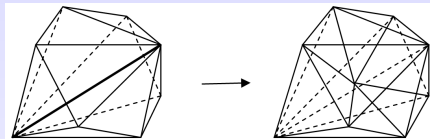
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$$Pe_0 = 80, \quad \tilde{\Gamma} = 40$$

Isotropic Adaptation: 3D

- In 3D the refinement loop is more complicated.



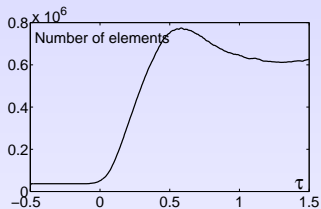
- The loop finishes if the macro-triangulation satisfies the Kossaczky condition (The mesh is derived from hexahedral triangulation and they are classified in 3 types of elements):

I.Kossaczky. Journal of Computational and Applied Mathematics (1994).

- 1 An arbitrary mesh needs a pre-adaptation divided each tetrahedron into twelve tetrahedra.
- 2 Use a mesh generator satisfying the required conditions:
R. Montenegro, J.M. Cascón, J.M. Escobar, E. Rodríguez, G. Montero. Institute for Intelligent Systems and Numerical Applications in Engineering (2009).

Isotropic Adaptation: 3D

Tridimensional simulations



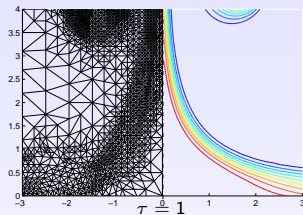
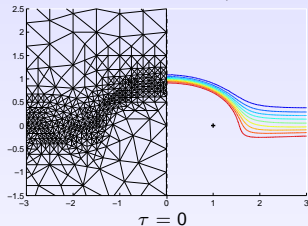
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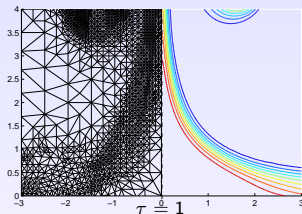
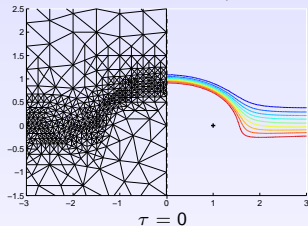
Anisotropic versus Isotropic Adaptation: 2D

- With the bisection algorithm mesh elements are adjusted only in size.
- In some cases the solution shows directional features.
- In these situations, anisotropic meshes might provide better results.

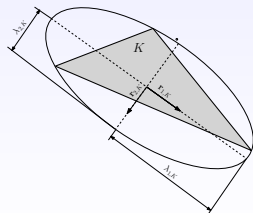


Anisotropic versus Isotropic Adaptation: 2D

- With the bisection algorithm mesh elements are adjusted only in size.
- In some cases the solution shows directional features.
- In these situations, anisotropic meshes might provide better results.



- The size but also the shape and orientation of the mesh must be defined. Metric tensor.



Anisotropic Adaptation: 2D

The metric tensor parameters depend on the a posteriori error estimator:

$$\|Z - I_K(Z)\|_{L^2(K)} \leq C |K| \left[\sum_{i=1}^d s_{i,K} (\mathbf{r}_{i,K}^T G_K(Z) \mathbf{r}_{i,K}) \right]^{1/2}$$

with $G(Z)$ Hessian matrix of the solution Z where $\lambda_{i,K} = s_{i,K} |K|^{1/d}$

Anisotropic Adaptation: 2D

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with $G(Z)$ Hessian matrix of the solution Z where $\lambda_{i,K} = s_{i,K} |K|^{1/d}$

- The optimal shape and orientation of the triangle are given by:

$$\hat{s}_{i,k} = \left(\prod_{i=1}^d g_{i,K} \right)^{1/d} g_{d+1-i,K}^{-1}$$
$$\hat{\mathbf{r}}_{i,K}^T = \mathbf{l}_{d+1-i,K}$$

where \mathbf{l}_i and $g_{i,K}$ is the eigenvector and eigenvalue of the Hessian matrix $G(Z)$.

- The size of the triangles is given by A_K^{opt} computed before.
- Mesh generator code used in this work:

F. Hecht, 'BAMG: Bidimensional Anisotropic Mesh Generator'

Anisotropic Adaptation: 2D

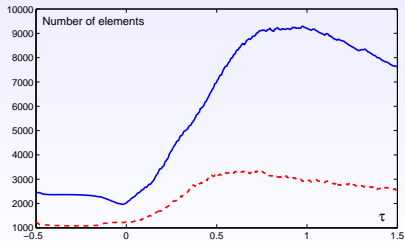
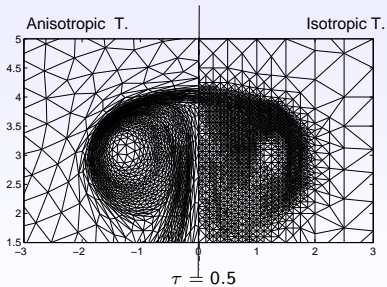
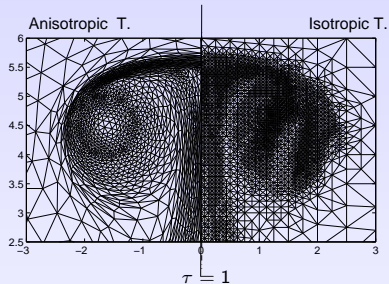
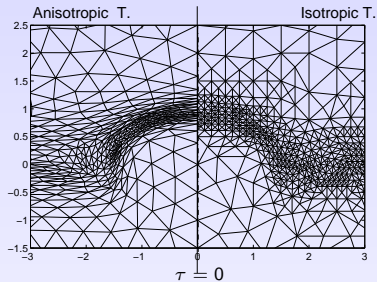
Simulation with anisotropic elements: Axisymmetric configuration

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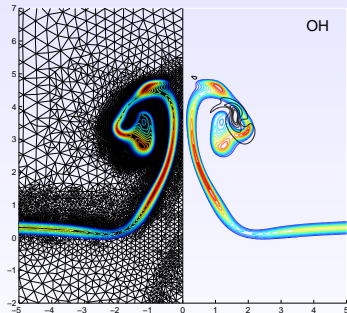
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$$Pe_0 = 80, \quad \tilde{\Gamma} = 40$$

Anisotropic Adaptation: 2D



- Model II (Complex model)
 - Detailed transport models.
 - H_2 /Air chemical kinetic model.



Model II: Constitutive Equations

$$\rho = \frac{M}{T},$$

$$M = \frac{1}{M_A} \left(\sum_{i=1}^I Y_i / M_i \right)^{-1}$$

$$\mathbf{q} = -\lambda \nabla T + \sum_{i=1}^I h_i \mathbf{j}_i,$$

$$\lambda = \frac{\lambda' T_0}{\rho_A D_{TA} h_A}, \quad h_i = h'_i / h_A$$

$$\mathbf{j}_i = \rho Y_i (\mathbf{v}_i^D + \mathbf{v}_i^T + \mathbf{v}_c),$$

$$\rho D_i = \frac{\rho' D'_i}{\rho D_{TA}}$$

Ordinary diffusion

$$\mathbf{v}_i^D = -\frac{D_i}{X_i} \nabla X_i$$

Thermal diffusion

$$\mathbf{v}_i^T = \frac{D_i \theta_i}{X_i} \frac{\nabla T}{T}$$

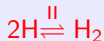
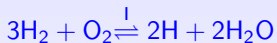
Correction velocity

$$\mathbf{v}_c = -\sum (Y_i \mathbf{v}_i^D + Y_i \mathbf{v}_i^T)$$

where h'_i , μ' , λ' , D'_i and θ_i are functions of the local thermodynamical state of the mixture (Kee et al., CHEMKIN (1983)).

Model II: Chemistry Model

$$\dot{m}_i = \frac{\dot{m}'_i}{\rho A A_{\text{ext}}}$$



Reaction		A	n	T_a [K]
1. $\text{H} + \text{O}_2 \rightleftharpoons \text{OH} + \text{O}$	k_f	$3.52 \cdot 10^{16}$	-0.700	8590
	k_b	$7.04 \cdot 10^{13}$	-0.264	72
2. $\text{H}_2 + \text{O} \rightleftharpoons \text{OH} + \text{H}$	k_f	$5.06 \cdot 10^4$	2.670	3166
	k_b	$3.03 \cdot 10^4$	2.633	2433
3. $\text{H}_2 + \text{OH} \rightleftharpoons \text{H}_2\text{O} + \text{H}$	k_f	$1.17 \cdot 10^9$	1.300	1829
	k_b	$1.29 \cdot 10^{10}$	1.196	9412
4. $\text{H} + \text{O}_2 + \text{M}_4 \rightarrow \text{HO}_2 + \text{M}_4$	k_0	$5.75 \cdot 10^{19}$	-1.400	0
	k_∞	$4.65 \cdot 10^{12}$	0.440	0
5f. $\text{HO}_2 + \text{H} \rightarrow \text{OH} + \text{OH}$		$7.08 \cdot 10^{13}$	0	148
6. $\text{HO}_2 + \text{H} \rightarrow \text{H}_2 + \text{O}_2$	k_f	$1.66 \cdot 10^{13}$	0	414
	k_b	$2.69 \cdot 10^{12}$	0.36	27888
7. $\text{HO}_2 + \text{OH} \rightarrow \text{H}_2\text{O} + \text{O}_2$		$2.89 \cdot 10^{13}$	0	-250
8. $\text{H} + \text{OH} + \text{M}_8 \rightarrow \text{H}_2\text{O} + \text{M}_8$	k_f	$4.00 \cdot 10^{22}$	-2.0	0
	k_b	$1.03 \cdot 10^{23}$	-1.75	59675
9. $\text{H} + \text{H} + \text{M}_9 \rightarrow \text{H}_2 + \text{M}_9$	k_f	$1.30 \cdot 10^{18}$	-1.0	0
	k_b	$3.04 \cdot 10^{17}$	-0.65	52092

San Diego Mechanism (2005)

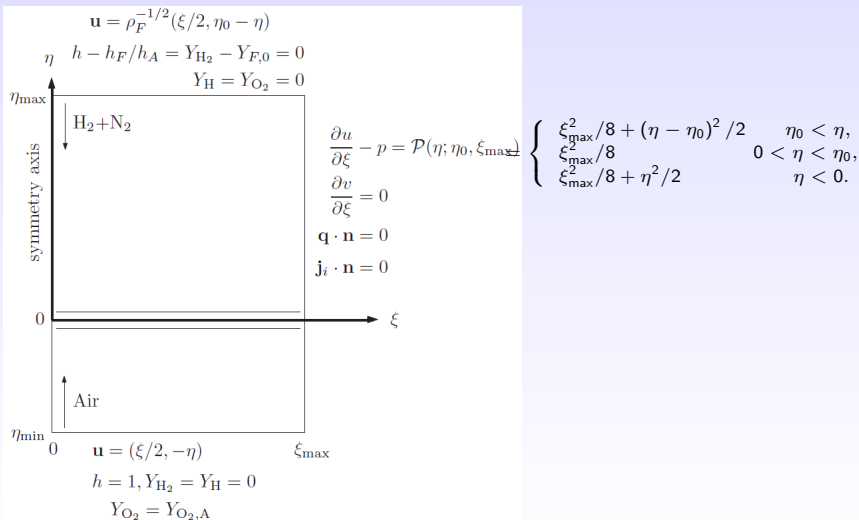
$$w'_I = k_{1f} C_{\text{O}_2} C_{\text{H}} - k_{1b} C_{\text{OH}} C_{\text{O}} + k_{5f} C_{\text{H}} C_{\text{HO}_2}$$

$$w'_{II} = k_{4f} C_{\text{M}_4} C_{\text{O}_2} C_{\text{H}} + k_{8f} C_{\text{M}_8} C_{\text{H}} C_{\text{OH}} + k_{9f} C_{\text{M}_9} C_{\text{H}} C_{\text{H}}$$

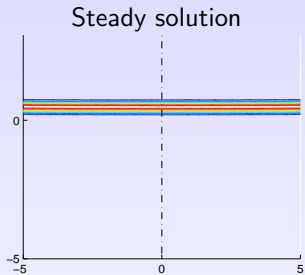
Mauss et al. (1993)

Model II: Boundary Conditions

Boundary conditions are taken from the steady unperturbed counterflow configuration



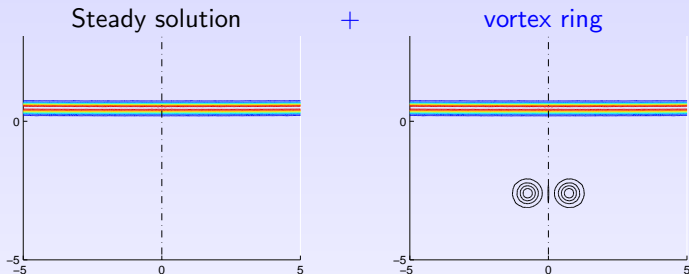
Model II: Initial Conditions



$$U = U_{SS}$$

$$V = V_{SS}$$

Model II: Initial Conditions



$$u = u_{ss} - \frac{\tilde{\Gamma}}{\pi \xi_c} \left(\frac{\xi_c}{2\xi} \right)^{3/2} \left(\frac{\eta - \eta_c}{\xi_c} \right) h_1(\mu) \left[1 - e^{-(\rho/\delta_v)^2} \right]$$
$$v = v_{ss} + \frac{\tilde{\Gamma}}{\pi \xi_c} \left(\frac{\xi_c}{2\xi} \right)^{3/2} \left[\frac{\xi}{\xi_c} h_1(\mu) - I_0(\mu) \right] \left[1 - e^{-(\rho/\delta_v)^2} \right]$$

$$\tilde{\Gamma} = \Gamma / (2A_0 r_0^2)$$

Hermanns et al., Combustion & Flame (2007)

Numerical Method: Compute the numerical solution

- Navier-Stokes and energy-mass conservation can be written as:

$$\rho \left(\frac{\partial c}{\partial \tau} + \mathbf{u} \cdot \nabla c \right) = \nabla \cdot \mathbf{f}(c) \quad \text{in } \Omega \times (\tau_{n-1}, \tau_n]$$

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- They can be uncoupled using a Semi-Lagrangian scheme to treat convective terms (only $\mathbf{u}^{n-1}, \mathbf{u}^{n-2}$ are needed)

$$\rho^* \frac{\partial c^*}{\partial \tau} = \nabla^* \cdot \mathbf{f}(c^*) \quad \text{in } \Omega \times (\tau_{n-1}, \tau_n]$$

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- ① Energy-mass equations are solved with an explicit Runge-Kutta Chebyshev (RCK) scheme in time (second order) and P_2 in space. h^n, Y_i^n and from them T^n .

Bermejo & Carpio, Appl. Num. Math. (2008)

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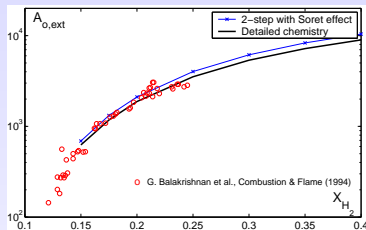
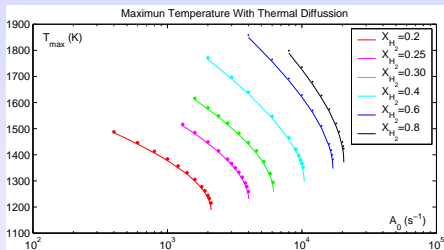
Bermejo & Carpio, Appl. Num. Math. (2008)

- 2 Navier-Stokes is solved with a BDF scheme of 2^o order in time and Taylor-Hood P_2/P_1 in space. \mathbf{u}^n and p^n .

- The error indicator η^n is measure for:

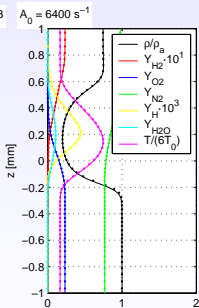
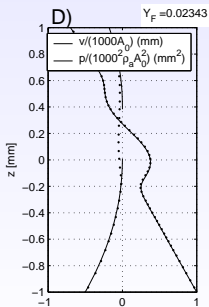
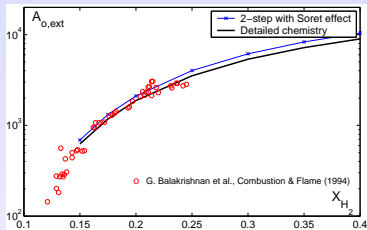
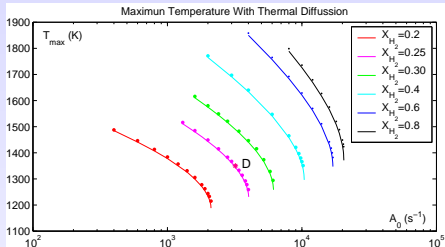
$$W(c) = 10^3 Y_H + 10^{-3}(u^2 + v^2)$$

Validation: Steady-State Extinction Curves

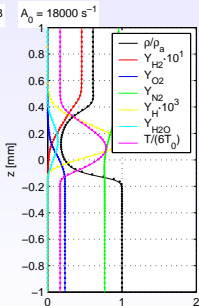
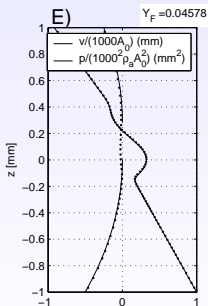
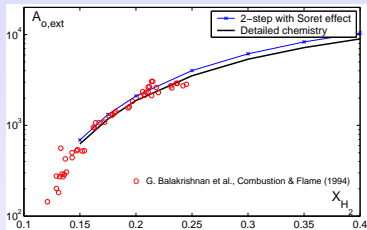
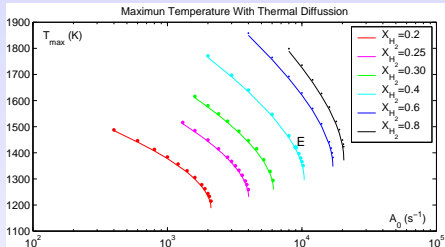


- We compute the steady-state counterflow configuration
 - The steady solutions are used as initial conditions for the unsteady calculations
 - The numerical solution is validated with COSILAB

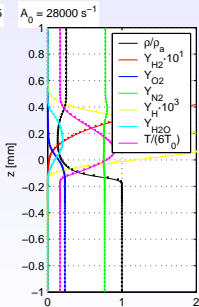
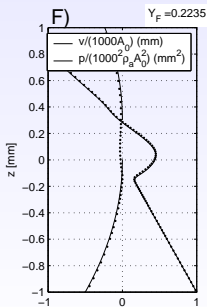
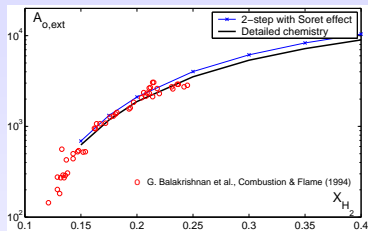
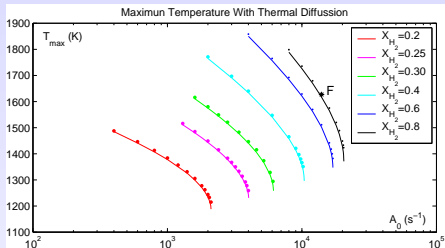
Validation: Steady-State Extinction Curves



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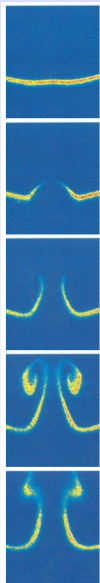


Validation: Steady-State Extinction Curves



Numerical Results: Case 1

Renard et al., Combustion & Flame (1999)

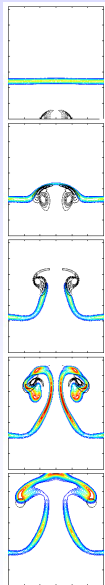
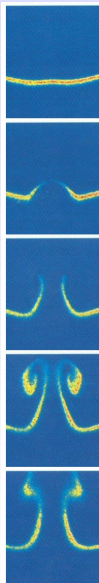


$$Y_{F0} = 0.015 \rightarrow A_{ext} = 1300s^{-1}$$

$$Pe_0 = 30, \quad \tilde{\Gamma} = 33, \quad \mathcal{R} = 30$$

Numerical Results: Case 1

Renard et al., Combustion & Flame (1999)



Carpio et al., 2011

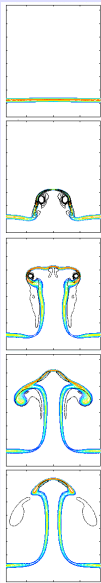
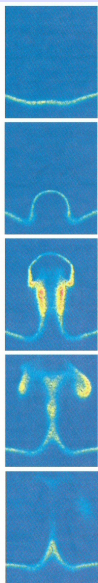
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$$Y_{F0} = 0.015 \rightarrow A_{ext} = 1300s^{-1}$$

$$Pe_0 = 30, \quad \tilde{\Gamma} = 33, \quad \mathcal{R} = 30$$

Numerical Results: Case 2

Renard et al., Combustion & Flame (1999)



Carpio et al., 2011

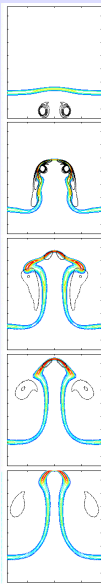
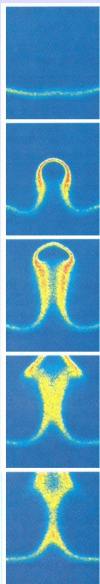
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$$Y_{F0} = 0.02 \rightarrow A_{\text{ext}} = 3200\text{s}^{-1}$$

$$Pe_0 = 60, \quad \tilde{\Gamma} = 35, \quad \mathcal{R} = 45$$

Numerical Results: Case 3

Renard et al., Combustion & Flame (1999)



Carpio et al., 2011

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$$Y_{F0} = 0.029 \rightarrow A_{ext} = 6000s^{-1}$$

$$Pe_0 = 40, \quad \tilde{\Gamma} = 40, \quad \mathcal{R} = 100$$

Conclusions

Conclusions

- A novel space-adaptive finite element algorithm has been used to simulate diluted hydrogen-air diffusion-flame/vortex interactions.
- Two different mathematical models have been used here.

Model I(Simple model)

- This model assumes constant physical coefficients and infinitely robust flames.
- This model allows us to analyze several refinement strategies. Anisotropic adaptation shows the better behavior.

Model II(Complex model)

- The mathematical model accounts detailed transport (CHEMKIM approach) and assumes a two-step reduced kinetic mechanism for hydrogen-air combustion.
- The results have been validated quantitatively using steady-state extinction curves and qualitatively against previously published experimental results.

Ideas to create an adaptive finite element method for combustion problems

Jaime Carpio Huertas

A jointly work with:

Professors:

Rodolfo Bermejo

Marcos Vera

Miguel Hermanns

Amable Liñán

Students:

Pablo Fraile

Ángel Fernández

Workshop on mathematical modelling of combustion
23-25/05/2011

